Development of Mechanism for Using in Multipurpose Wheelchair Using Dimensional Synthesis of Mechanism

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Abstract - The wheelchair is transportation device for patients, physically handicapped people, and old age people. These wheelchairs have been using for many decades. As the technology is developed with time, much advancement took place in the design of the wheelchair. Though there is much advancement, the patients are still facing some problems with the wheelchairs while flipping wheelchair into bed. This is the situation happening for many years. So, to avoid this problem and make them ease to convert their wheelchair directly to the bed whenever they require, this mechanism will help them. This mechanism makes the patient very comfortable in converting his/her own wheelchair into bed. This multipurpose wheelchair is developed by one of the methods in dimensional synthesis of mechanism which is "FUNCTION GENERATION-THREE POSITION-Graphical method" to define the dimensions and positions of the links.

Index Terms - Wheel chair, Multipurpose wheelchair, Dimensional synthesis, function generation

I.INTRODUCTION

The wheelchair is designed using the mechanisms which are the combination of rigid links. So that the parameters like velocity and acceleration are under control as of our requirement. This makes the patients to operate the chair very easily. Such mechanism used in this multipurpose wheelchair is developed by using dimensional synthesis of mechanism. The chair can be adjusted for both sitting and sleeping positions.

In synthesis of this mechanism, by using one of the methods of dimensional synthesis of mechanism that is "FUNCTION GENERATION-THREE POSITION-Graphical method" the dimensions and positions of the links are defined.

II.PROCEDURE FOR MECHANISM DEVELOPMENT BY FUNCTION GENERATION-GRAPHICAL METHOD

The fixed link, input link and output link are taken as per the requirement.



Figure.01: Links taken as of our requirement ABC-Fixed link, CD1-Input link and BE, AF-Output links

AB= 4 units, AF=4 units, BE=4 units, pC=1 unit, CD1=1 unit.

Relationship between input and output links (as per the requirement)

 $\theta_{12}=45^{0}, \theta_{23}=45^{0}$ $\alpha_{12}=45^{0}, \alpha_{23}=45^{0}$ $\beta_{12}=37.5^{0}, \beta_{23}=37.5^{0}$ $\theta=$ input link position $\alpha, \beta=$ output links position



Figure.02: Relationship between input and output links



1. Draw a line joining BD₂, with B as center locate D^{1}_{2} with an angle α_{12} =45⁰ counter clock wise from D₂.





2. Draw a line joining BD₃, with B as center locate D_3^1 with an angle α_{13} =90⁰ counter clock wise from D₃.



Figure.04: Step1-(ii)

3. Now join D_1 , D^1_2 , and D^1_3 , draw perpendicular bisectors to $D_1D^1_2$ and $D^1_2D^1_3$, locate intersection point as G.



Figure.05: Step1-(iii)

 Join E, B, and G. EBG-rigid link (output link). Join D₁ and G. D₁G-rigid link (coupler).



STEP: 2

1. Draw a line joining AD2, with A as center locate D112 with an angle β 12=37.50 counter clockwise from D2.



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AH=1.2 units Angle (FAH)=56.63⁰

E

Figure.11: Mechanism in sitting position

1.Sleeping position:

Whenever the input link rotates, the two-output links moves i.e., 90° clockwise rotation of CD₁ gives the 90° clockwise rotation of BE and 75° clockwise rotation of AF.



Figure.12: Mechanism in sleeping position

IV. RESULT

Dimensions of the links are: a. Output link 1-EBG: EB=4 units BG=1 unit Angle (EBG)=135⁰



 $\label{eq:Figure.14: Output link-2} Figure.14: Output link-2 a. Coupler 1-HD_1: HD_1=2.07 units b. Coupler 2-DG: D_1G=2.23 units$

Mechanism in three positions: 1. When $\theta=0$

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Figure.15: Mechanism when $\theta=0$

2. When $\theta_{12}=45^{\circ}$



Figure.16: Mechanism when $\theta_{12}=45^{\circ}$

8 = 90

3. When $\theta_{13}=90^{\circ}$



Figure.17: Mechanism when θ 13=900

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