

# An Effect of Different Copper Fins and Internal Grooving of Air Cooled Counter to Cross Flow Heat Exchanger - A Review

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**Abstract-**The objective of this study is to provide modern analytical and empirical tools for evaluation of the thermal-flow performance or design of air-cooled heat exchangers (ACHE) with copper material of rectangular fin shape and cooling towers. This review consists various factors which effect the performance of ACHE. A fan will be fitted perpendicular to the tube which expand cold air through the fan on tube. When the hot fluid will be flow in tube then internally groove shaped of tube due to high turbulence in fluid. The performance of the various types of fins is presented with respect to the parameters: (1) Reynolds number, (2) fin pitch, (3) fin height, (4) fin thickness, (5) tube diameter, (6) tube pitch, (7) tube type, (8) number of tube rows, and (9) effect of dehumidifying conditions. .The total cost of cycle can be reduced by increasing the effectiveness of heat exchanger. In this Section the conclusions and the recommendations for the future work have been given.

**Keywords-** Internal grooving, rectangular fin, pressure drop, Annular tube, heat transfer coefficient, turbulent flow

## 1. INTRODUCTION

A heat exchanger is a device that is used to transfer thermal energy (enthalpy) between two or more fluids, between a solid surface and a fluid, or between solid particulates and a fluid, at different temperatures and in thermal contact. In heat exchangers, there are usually no external heat and work interactions. Typical applications involve heating or cooling of a fluid stream of concern and evaporation or condensation of single- or multicomponent fluid streams. In other applications, the objective may be to recover or reject

heat, or sterilize, pasteurize, fractionate, distill, concentrate, crystallize, or control a process fluid. In most heat exchangers, heat transfer between fluids takes place through a separating wall or into and out of a wall in a transient manner. In many heat exchangers, the fluids are separated by a heat transfer surface, and ideally, they do not mix or leak. Such exchangers are referred to as direct transfer type, or simply recuperators. In contrast, exchangers in which there is intermittent heat exchange between the hot and cold fluids—via thermal energy storage and release through the exchanger surface or matrix—are referred to as indirect transfer type, or simply regenerators.

A heat exchanger consists of heat transfer elements such as a core or matrix containing the heat transfer surface, and fluid distribution elements such as headers, manifolds, tanks, inlet and outlet nozzles or pipes, or seals. Usually, there are no moving parts in a heat exchanger; however, there are exceptions, such as a rotary regenerative exchanger (in which the matrix is mechanically driven to rotate at some design speed) or a scraped surface heat exchanger. To increase the heat transfer area, appendages may be intimately connected to the primary surface to provide an extended, secondary, or indirect surface. These extended surface elements are referred to as fins. Thus, heat is conducted through the fin and convected (and/or radiated) from the fin (through the surface area) to the surrounding fluid, or vice versa, depending on whether the fin is being cooled or heated. As a result, the addition of fins to the primary surface reduces the thermal resistance on that side and thereby increases

the total heat transfer from the surface for the same temperature difference. Fins may form flow passages for the individual fluids but do not separate the two (or more) fluids of the exchanger. These secondary surfaces or fins may also be introduced primarily for structural strength purposes or to provide thorough mixing of a highly viscous liquid.

Types of air-cooled heat exchanger [ACHE]:

- Induced draft ACHE
- Forced draft ACHE
- Natural draft ACHE
- Forced draft ACHE - In air cooled heat exchanger the fan is situated underneath the procedure pack and air is constrained through the tubes.

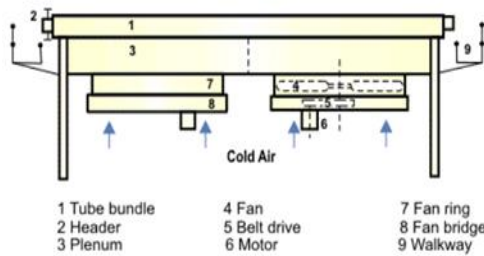


FIG.1- Forced Draft ACHE

- Induced draft ACHE - In air cooled heat exchanger the fan is situated over the procedure pack and air is pulled, or prompted, through the tubes

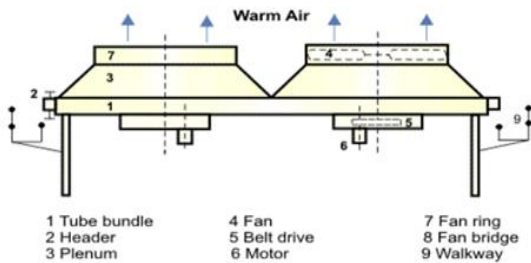


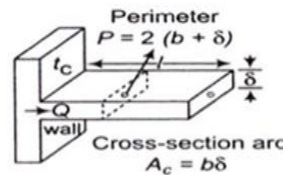
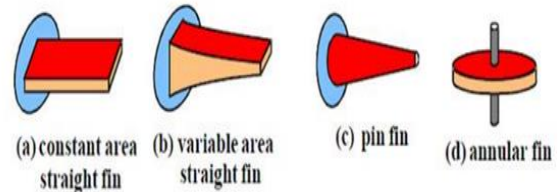
FIG.2-Induced Draft ACHE

- Natural draft ACHE- Natural draft air cooled heat exchanger is synonymous. There are no used fans to drive air. In the natural draft air drive through the tubes group. Its logic is comparable to induced draft.

**FINS:** In analysis of heat transfer, fins are device which are used to enhance the rate of heat transfer of surface that expand from things to increases the rate of heat transfer. The rate of heat transfer depends on: Shape and geometries of the outer surface, the Surfaces area of body, velocity of the any fluids in other case, temperature of surroundings etc. There is

various type of fins applications to enhance the rate of heat transfer from the surfaces. Which fin are used to be enhance the rate of heat transfer of surface the fin material should be higher thermal conductivity. The blade is presented to a streaming liquid, which cools or warms it, with the high warm conductivity permitting expanded warmth being directed from the divider through the balances.

There are various type of fins:



Rectangular Fin Diagram

Such as there are various type of the fins but we will be used constant area straight fin [rectangular fin] because its surface are equal due to heat transfer are equal at every point of the fins surface due to enhances the rate of heat transfer of the surface. Such as rectangular fins will be used in air cooled heat exchanger because rectangular fins material will be copper due to its thermal conductivity are high. In air cooled heat exchanger when hot fluid will be flow in GI tube than the rectangular copper fins are brazing on the GI tube due to rectangular copper fins will be the heat transfer rates will be increased on the surface of tubes.

## 2.LITERATURE REVIEW

Bayram Sahin et al. [1] In this investigated paper the design parameter of a heat sink on which hollow trapezoidal baffle is mounted on the base surface. This experimental design is used Taguchi method. Where Nussle numbers and friction factors are considered as performance parameter. In which orthogonal arrays are selected as investigational plan for the six parameters: the curve angles( $\alpha$ ), the inclination angle ( $\beta$ ), the baffle heights(H), the baffle lengths(L), the

baffle width(S) and Reynolds numbers. First of each goal has been optimized individually and after that all the goals have been optimized together. The baffle lengths(L) are found on the friction factor. The baffle length will in flow direction. Where are the best parameters on the exchange warmth is Reynolds number. The result showed that the heat transfer was obtained at  $Re=17,000$ ,  $H=36$  mm,  $L=45$  mm,  $S=26$  mm,  $\alpha=0^\circ$ ,  $\beta=0^\circ$ . It can be conclude higher heat transfer rates are achieved with lower pressure drop.

Jie Qu et al.[2] This research paper discussed the heat transfers of PCM and enhance the thermal performances of PCM used two type of novels are 3D-OHPs (4 layers 3D-OHP and 3 layers 3D-OHP) and PCM coupled with multiple 2D-OHPs. Phase change materials (PCM) have been mostly use in thermal managements because its have high latent heat and low price . Phase change materials are use Oscillating heat pipes (OHP) and effective thermal transfer devices for heat transfer. Its due to enhance the thermal performances. In this experimental manuscript both novels 3D-OHPs and regular OHPs are hired for the PCM thermal performance. By knowing this research paper Results shows that paraffin wax/3D-OHP systems are needed extra times for finally melting of paraffin wax than paraffin wax/OHPs system. In the solidification process two systems are performed better than simply paraffin wax. The solidification times of the pure paraffin wax and paraffin wax/4-layers 3DOHP are taken only 0.29 times whereas the paraffin wax/4 OHP systems are 0.48 times taken. Such as the paraffin wax/3D-OHP systems are superior performances.

Hassan Jafari Mosleh et al.[3] This novels are focused on experimental and numerical investigation of pulsating heat pipes(PHPs) as substitutes for fins in a representative air-cooled heat exchanger(ACHE). Because of low temperatures difference between the cooling air and internal airflow. In which R134a was selected as the best working fluid from the heat transfer standpoint. When heat transfer has been done by conduction throughout the wall of the PHP-tube. Than PHPs are filled with working fluid, the coefficient of heat transfer and temperature difference have been increased. In this condition the performances of the PHP-tubes are noted without working fluids are similar to the fin tube. When the axial fans are stopped due to small gap between the fins and produce poor thermal performances of the fin.

By knowing this research paper results shows that using PHPs instead of fins improves heat transfer efficiency. In which Firstly fins and PHP-tubes are tested without any exterior flow over the main tubes and the tests are conducted in natural convection situations.

Jian Wang et al. [4]In this research paper experimentally study of the heat transfers and flow individuality of the three new finned copper head heat sinks are subjected to the impingement chilled by rectangular slot jet and axial fan. These experimental processes are used for the fast development of electronic devices has imposed higher requirements for thermal supervision and cooling technology. In this experiment taken effect of heat sink heights (H, 15, 30, 45, 60 mm), the pore density of the inserted copper head (PPI, 10, 20, and 30) and the gas flows Reynolds number( $Re$ , varying from 2053- 12737) are scientifically investigated. Where are two types of conventional fin heat sink with 8 and 22 fins but without copper heads are tested for judgment. Such as Experimental results expose that inserting copper heads are completely enhance the thermal performances of finned heat sinks. Finned copper foams and conservative heat sinks with the same numbers of fin but finned copper foams are better heat transfer performance. By knowing this experimental research paper, when the height of heat sinks decreases than pressure drop for all five kinds of heat sinks increases.

De-Shau Huang et al.[5] In this investigate paper experimental and numerical simulations of 30W LED automotive headlights using for heat indulgence. In this experiment, we required to enhance the efficiency of heat transfer of LED take on fins with a grooved heat pipe on the heat sinks. ANSYS Fluent software is used just before model of the heat transfer mechanisms and aid in the design of heat pipes and heat sinks. The temperature distribution of the LED headlights are computer-generated for various material of the heat sinks and printed circuit board (PCB), and fins are designed with a heat pipes. It was found increases the coefficient of thermal conductivity of the substrate due to decreases in the LED junction temperatures, however higher thermal efficiency was not essentially enough. In this trial demonstrated that the reconciliation of 76-mm-since quite a while ago furrowed warmth channels with a compelling warm conductivity of  $6000$  W/(m•K) and 2-mm arduous

plate heat dissemination balances on the warmth sink with an AlN Ceramic having a  $180 \text{ W}/(\text{m}\cdot\text{K})$  demonstrated successful in scattering heat from powerful LED headlights inside a profoundly constrained space. By knowing this research paper enhance the coefficient of thermal conductivity of the substrate due to results are decreases in the LED junction temperature. However the higher thermal efficiency are not necessarily sufficient.

Hai Wang et al.[6] This investigated papers are thermal performance of the oscillating heat pipes(OHP) designed, temperature distribution and explanation profiles of LED array are experimentally tested and evaluated. In this experiment the thermal management of high-powered LED chips are designed and fabricated, where are the tubular oscillating heat pipes(OHP) with sintered copper particles(SCPs) are inside of the flatplate evaporator. The thermal performances of designed OHP, temperature giving out and explanation profiles of LED array were experimentally tested and evaluated. A low substantial ratio of 30% are preferred for the designed OHP practical in high-power LED cooling. By knowing this research paper, heat sink of tubular OHP with SCPs inside the flat plate evaporators are developing for the cooling of high-power density array. The performances of LED heat sinks are experimentally investigation of the effect of evaporators with SCPs, power input, inclination angle and filling ratio. When the addition of the sintered copper particles (SCPs) with oscillating heat pipes (OHP) due to appreciably enhance the vapor bubble generation rate. The temperature division of the LED array at input power and low filling ratio have different inclination angles are instant or less than  $70 \text{ }^\circ\text{C}$ . The filling ratio affect the OHP put in place performance. The low filling ratio are ideal for the OHP practical because the thermal management of high-power LED. In this experimental setup, the temperature of LED array is inversely proportional to the explanation intensity.

Demis Pandelidis et al. [7] in this research relative study of the sloping evaporative exchangers are worked as heat recovery units, given configurations are counter flow and cross flow. In this experiment presented analysis are accepted out with particular importance on the condensation process that occurs in the product air channels of the exchangers. In which various aspects are related to the water vapor condensation and manage. Which aspects are taken in

the classification factors that control the condensation process. Those analyzing factors are forced on dissimilar IEC exchanger arrangement. There are a variety of inlet parameter and operating condition for judgment of the counter and cross flow exchangers. Those performed analysis are based on numerical simulations with mathematical e-NTU models of heat and mass transfer. By knowing this research paper found result showed that the counter flow configuration has high reasonable and latent cooling potential than the cross flow unit. The technical limitation of counter-flow configuration due to the cross flow exchangers are achieved higher Energy Efficiency Ratio and lower investment cost. Whenever, the structure of the counter-flow design that requires supplementary input or output branch of the results are increased size of the counter flow components and higher pressure drop balance with the cross flow exchangers. Lei Wang et al. [8] In this study paper author focus on the article in presented a new mathematical model that combined the law of energy conservation and the principle of the irreversible thermodynamic theory. In this investigation the wet bulb indirect evaporative cooling (IEC) achieved through M-Cycle is a difficult thermodynamic process. Heat and mass transfer for advance understanding occurs in a dew point indirect evaporative air cooler with M-Cycle counter flow configuration. The research paper is represented mathematical model. The model comprising of various energy, mass and entropy equations are uses to take out the study of the dew point air cooler below various operational and structural conditions. The entropy creation numbers are establish to be a show potential indicator for the optimized designs. The mutual analysis are energy efficiency and thermodynamic irreversibility of the intention IEC system. The optional average air velocities of dry channels should be less than  $1.0\text{m/s}$ . In this experimental setup the channel length should be in range of  $1\text{e}1.75\text{m}$  and channel gap should be controlled to  $3\text{e}5 \text{ mm}$ . Whereas the working to intake air ratio must be around  $0.3\text{e}0.4$ . By knowing this research paper, for better and advance understand of the heat and mass transfers are occurred in a dew point indirect evaporative air cooler with M-Cycle counter flow configuration. Based on this experimental study has been proved that the entropy production numbers are used for the useful parameters

in the optimization designs of the HMX for a dew point IEC.

Anna Pacak et al. [9] In this study paper, the investigation of the heat and mass transfers in counter flow heat exchangers are performing on the foundation of organized to order  $e - NTU$  model. The rate of specific heat and mass transfers zones was conventional for different exhaust airflow parameters and the heat exchanger efficiency. The local reasonable and latent heat transfers rate is sharing out analyze for the different values of heat exchanger efficiency. The experimental investigations are approving away to confirm the original mathematical model of plate heat exchangers. The successful results are judgment between effective and experimental data showed that the numerical models are accomplished to calculate effectively of the plate heat exchangers operating performance. There are two freeze safety method, for example preheating and bypassing of the outdoor airflow are taking into reflection. By knowing this research paper, Bypass freeze protection method is characterize by low heat improvement efficiency for the a large amount characteristic judgment range of the return air relative humidity  $RH_{2i} \frac{1}{4} \delta 20 . . . 50\%$ . Such as Bypass freeze protection method provides higher total power demand compared to the preheating method. The problems of water air heater freezing are done through the bypass method treatment. The investment cost of bypass methods are higher than the preheating method. Bypass method are taken into accounts as soon as makes final decision.

Ali Pakari et al.[10] This research paper are the mathematical model of the heat and mass transfers procedure in counter flow dew point evaporative cooling systems are developed. The measured input prepared parameters are inlet air temperature, inlet air velocity, inlet air relative humidity, and extraction ratio. Which measured geometrical parameters are the canal length and canal width of the cooling system. The chosen output responses are outlet air temperature, outlet air relative humidity, and wet-bulb efficiency. These experimental measurements are based on numerical model. The input factors are inlet temperature, inlet relative moisture, inlet velocity, removal ratio, canal length, and canal width. The output response of the cooling system are exit temperature, exit virtual moisture, and wet bulb efficiency. There are using the CCD, second order degeneration models are built-in to the selected

responses. The degeneration models speak about the six input factors to the three selected responses. The degeneration models used for forecast of the presentation and the optimization of the aim of counter flow dew point evaporative cooling systems.

Xia Song et al. [11] In this research paper logical methods and simulation of work has analyzed the association between the temperature levels of heating and cooling source and the air state in a typical LD systems regenerated by arrival air. The circulating solutions are the media to transfer the heat and mass. Where are the transfer heat and mass between the air heating and cooling sources. The logical results show that certain air states (clean air, supply air, and return air) keep in touch to one particular solution circulation. Such as the certain air can only be expert by a certain pair of temperatures for heating and cooling sources. The supplied air has additional influenced in the last with regard to the temperature of heating and cooling sources. By knowing this research paper at a set temperature, a minor humidity ratios of the supplied air demand, A momentous enhances in the temperature of heating sources and minor decreases in the temperature of cooling source. The lower temperature of supplied air require low temperature for both source on fixed humidity ratio. The moisture ratio of the clean air is the main factor that determine the temperature of two sources. The dehumidifier is the very significant component and its outlet air is unwavering by the inlet solution less than a confident inlet air, in step heat capability flow rates and a confident NTU m.

Mirco Rampazzo et al.[12] This research, measured an Indirect Evaporative Free Cooling System. The experiment is based on mathematical modelling and given transmission of explore in shorter time and smaller costs. In this term paper first, we grow a First Principles data driven models for an Indirect Evaporative Cooling system with Free Cooling and then we consequently design a mat labs based process environment. Evaporative Cooling and Free-Cooling technology have obtain a increasing interest in air-conditioning systems and they are appropriate in different air conditioning applications: commercial, industrial, residential, and data centers. Such as, the Evaporating Cooling technology is eco-friendly and this technology could offer tremendous cooling and exposure to air through least energy consumptions. Evaporative Cooling technology has a very little global warming impacts. By knowing this research

paper, we have given attention an Indirect Evaporative Free Cooling System. The main aspects of an Indirect Evaporative Free Cooling System are heat exchanges, air and water temperatures, etc. The physical phenomena and experimental datas are developed by the First-Principle Data-Driven model. The fixed stirring boundary methods for telling the input constituents of the system, i.e. the heat exchangers, have provided a good quality balances between model's complexity and accuracies.

### 3. OBJECTIVES

After study of different research paper, I understood about different types of grooved pipe, fins, air cooled heat exchanger configurations using is presented by different authors. By Bayram Sahin research.

- In this Optimization of intend parameter for heat transfer and friction factors in heat sinks with hollow trapezoidal baffle exploratory investigation, the impact of various structure parameters on warmth move and weight drop qualities in a warmth sink furnished with trapezoidal astounds were examined by applying Taguchi trial plan strategy.
- In this research papers experimental investigations on thermal performances of phase changed materials attached with three-dimensional oscillating heat pipe (PCM/3DOHP) for thermal management application analysis by Jie Qu.
- In this paper, two sorts of frameworks (PCM/3D-OHP and PCM/ OHPs framework) for warm administration application have been made and tried during liquefying and cementing process. By author Anna Pacak investigation of power demand calculations to freeze prevention method of counter flows heat exchanger used in energy revival from exhaust air.
- In which state that a hypothetical examination of heat and mass exchange in a counter-stream heat exchanger utilizes for energy retrieval in air taking care unit (AHU) under below zero working conditions is displayed. In other follow a line of investigation by author Jian Wang investigate the heat transfer and flow characteristic in fin copper foam heat sink subjected to jet impingement cooling.

- In this research states that the heat move and weight drop of five sorts of warmth sinks showing to impingement cooling by rectangular space flow and fundamental fan. From the above research paper, various authors explained the operation on different tube and fins material and tube in different grooving profile with different fluid flow.

### 4.EXPECTED OUTCOME

When I study of various research paper of many others heat exchangers based than I found that different type of fins and pipe profile used that have different results.

- In the various research paper I had been study that parallel flow, counter flow and cross flow of heat exchanger in which different material of tubes and fins are used. In the tubes different profile such as circular, trapezoidal grooves found.
- So after study of different research paper of heat exchangers based I understood idea that research paper In which some profile on which works are not completed than I had been found that new ideas on which works are possible.
- After study of various research papers related to heat exchanger I decide that I will be works on counter to cross flow in air cooled heat exchangers [ACHE].
- In which fins are used copper for best thermal conductivity, pipe grooved profile is spiral groove for high turbulence and pipe material are used aluminium because its thermal conductivity is good and much preventive of corrosion.
- Thus after various research paper study, we found an idea and decide research on counter to cross flow in concentric tube air cooled heat exchanger [ACHE]

### REFERENCE

- [1] Afzal, A., Mohammed Samee, A.D., Abdul Razak, R.K "Optimum spacing between grooved tubes: An experimental study', Journal of Mechanical Science and Technology, Vol 34, No. 1, January 2020.
- [2] Nithiyesh Kumar, C., Ilangkumaran, M., "Experimental study on thermal performance and exergy analysis in an internally grooved tube integrated with triangular cut twisted tapes

- consisting of alternate wings”, *Heat and Mass Transfer*, Vol 55, April 2019.
- [3] Zhisong Li. “Design and preliminary experiments of a novel heat pipe using a spiral coil as capillary wick”. *International Journal of Heat and Mass Transfer* Volume 125, November 2018.
- [4] Pengxiao Li, Peng Liu, Zhichun Liu, Wei Liu “Experimental and numerical study on the heat transfer and flow performance for the circular tube fitted with drainage inserts”. *International Journal of Heat and Mass Transfer* Volume 107, April 2017.
- [5] Pankaj N. Shrirao, Rajeshkumar U.Sambhe, Pradip R.Bodade, “Convective Heat Transfer Analysis in a Circular Tube with Different Types of Internal Threads of Constant Pitch”. *International Journal of Engineering and Advanced Technology (IJEAT)*, Volume2, Issue-3, February 2013.
- [6] Kadir Bilen, Murat Cetin, Hasan Gul, Tuba Balta, “The investigation of groove geometry effect on heat transfer for internally grooved tubes”. *Applied Thermal Engineering*, Volume 29, Issue 4, March 2009.
- [7] P. Bharadwaj, A.D. Khondge, A.W. Date, “Heat transfer and pressure drop in a spirally grooved tube with twisted tape insert” *International Journal of Heat and Mass Transfer*, Volume 52, Issues 7–8, March 2009.
- [8] M. Siddique, A.-R. A. Khaled, N. I. Abdulhafiz, and A. Y. Boukhary, “Recent Advances in Heat Transfer Enhancements: A Review Report”, Hindawi Publishing Corporation, *International Journal of Chemical Engineering*, Volume 2010, September 2010.
- [9] S Basavarajappa, G Manavendra and S B Prakash, “A review on performance study of finned tube heat exchanger”, *Journal of Physics: Conference Series*, Vol. 1473, No. 1, February 2020.
- [10] M Goto, N Inoue, N Ishiwatari, “Condensation and evaporation heat transfer of R410A inside internally grooved horizontal tubes, Vol. 24, Issue 7, July 2001.
- [11] M. Goto, N. Inoue, R. Yonemoto, “Condensation heat transfer of R410A inside internally grooved horizontal tubes”, *Int. J. Refrig.* 26 (2003) 410–416.
- [12] V. Zimparov, “Enhancement of heat transfer by a combination of three-start spirally corrugated tubes with a twisted tape”, *Int. J. Heat Mass Transfer* 44 (3) (2001) 551–574.
- [13] P.G. Vicente, A. Garcia, A. Viedma, “Experimental investigation on heat transfer and frictional characteristics of spirally corrugated tubes in turbulent flow at different Prandtl numbers”, *Int. J. Heat Mass Transfer* 47 (4) (2004) 671–681.
- [14] P. Naphon, M. Nuchjapo, J. Kurujareon, “Tube side heat transfer coefficient and friction factor characteristics of horizontal tubes with helical rib”, *Energy Convers. Manag.* 47 (18–19) (2006) 3031–3044.
- [15] AGS Roll Groove Specifications for 14-72”/DN350-DN1800 roll groove dimensions, © 2019 Victaulic Company.